

# Re-Conceptualization and Optimization of a Rapidly Deployable Floating Causeway

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## ABSTRACT

There is an increasing demand for rapidly deployable causeways that can provide access from ship to shore for military and disaster relief operations. Existing systems have major limitations including only being transportable and emplaceable by large strategic sealift vessels, having high weight and packaged volumes, and requiring intensive on-site assembly. In response to the demand for a lightweight, air-liftable, quickly emplaceable causeway, the Engineer Research and Development Center has developed a prototype comprised of aluminum modules joined by compliant connections and supported by pneumatic floats. As research and development progressed and experience was gained, eliminating the heavy and complex compliant connections was identified as a potential improvement. To eliminate these compliant connections, the authors have re-conceptualized this design so that a desired superstructure flexibility (that takes advantage of buoyancy while meeting deflection limits) is achieved. The superstructure has been designed for a target stiffness to permit a desired curvature under a design moment. This paper will (1) review existing causeways, (2) present this re-conceptualization, and (3) discuss the optimization implemented to achieve this new design.

**CE Database subject headings:** Bridges; Floating structures; Military engineering; Composite material; Optimization

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## 21 INTRODUCTION

22 Causeways, meaning bridging systems capable of transporting troops and supplies from ship  
23 to shore, are a critical military asset to reach austere or damaged sites. The Department of De-  
24 fense has identified a strategic interest in reaching these sites by shallow-draft vessels with rapidly  
25 deployable causeways for offloading operations or disaster relief (Deming, 2009). Existing cause-  
26 ways can provide access to these damaged and small ports, but have the major limitation of only  
27 being transportable and emplaceable by deep-draft vessels with high-load capacity cranes. Fur-  
28 thermore, existing systems have a high weight and packaged volume, require intensive in-water  
29 assembly with substantial support equipment, and cannot be transported by air (Fowler et al., 2006;  
30 Deming, 2009). Toward this end, a prototype was developed by the Engineer Research and De-  
31 velopment Center (ERDC), known as the Lightweight Modular Causeway System (LMCS, Figure  
32 1), with technical support from Alion Science and Technology, Demaree Inflatable Boats, Ocea-  
33 neering International Incorporated, and Quantum Engineer Design Incorporated (Deming, 2009).  
34 This system is a modular, floating causeway comprised of aluminum modules joined by compliant  
35 connections and supported by pneumatic floats.

36 The existing prototype demonstrates a reduction in packaged volume and self-weight by at  
37 least 50% compared to other existing systems. A 36.6 m LMCS causeway can be shipped in the  
38 footprint of three International Standards Organization (ISO) freight containers (Deming, 2009),  
39 but the system is still somewhat heavier than the developers would like. The compliant connections  
40 between modules are fairly complex and make up a large component of the system self-weight.  
41 Toward this end, the authors have re-conceptualized this system such that the connections can  
42 be simplified and the required rotational compliance is achieved in the module instead. This is  
43 accomplished by designing a cross section with a target stiffness to permit a specific rotation under  
44 a given load. A box girder profile is assumed. The design of this section is performed using  
45 multi-objective structural optimization to minimize the self-weight of the section while reaching  
46 the target stiffness. This procedure is performed first for a glass fiber reinforced polymer (GFRP)  
47 material to explore opportunities for the use of advanced composites for lightweight design. An

48 analogous procedure is also performed to design optimized cross sections in two grades of high  
49 strength steel. All three resulting designs are compared against the existing prototype.

50 This paper will first briefly review the history of causeway systems developed by the US mil-  
51 itary, focusing on its newest design. Then it will discuss the desired re-conceptualization of the  
52 superstructure assembly. Next, the multi-objective optimization process will be discussed, includ-  
53 ing a review of relevant work in the field, a detailed problem formulation, and a description of the  
54 algorithm employed - Simulated Annealing (SA). Finally, the optimized designs will be presented  
55 and compared against the existing prototype.

## 56 **A BRIEF REVIEW OF CAUSEWAY SYSTEMS AND THE EXISTING PROTOTYPE**

57 To better understand the context for this re-conceptualization, it is critical to review (1) the  
58 history of deployable and rapidly erectable causeway systems starting after World War II and  
59 leading up to the current effort and (2) the existing prototype, including its advantages over prior  
60 systems and potential areas for improvement.

### 61 **History of Causeway Systems**

62 The history of deployable and rapidly erectable causeway systems begins with a temporary  
63 harbor that was used by the Allied forces during World War II - the Mulberry Harbour. Although  
64 the system proved invaluable until it was destroyed by a large storm, the harbor system was not  
65 replicated (Potts, 2009). Later systems were designed to be more transportable, less challenging  
66 logistically, and are briefly presented below.

67 Two of the earliest fielded causeway systems were the Navy Lighterage System (NLS) in the  
68 1960's and the Modular Causeway System (MCS) in the 1980s. Both consist of steel modular  
69 sections. The modules of the NLS are 6.40 m wide by 27.43 m long. Since they are so large,  
70 the modules require special lifting equipment to be put in place and they are not ISO compatible  
71 (able to be handled by ISO material handling equipment and transported by ISO compatible vehi-  
72 cles and trailers) (Garala, 2004; Anon., 2012). The MCS, which was adopted by the U.S. Army,  
73 was designed to be ISO compatible and can be configured in several different ways: the Floating  
74 Causeway, the Roll-On/Roll-Off discharge facility, the Causeway Ferry, and the Warping Tug. All

75 can be interchanged and connected both side-to-side and end-to-end (Buonopane, 2002; Depart-  
76 ment of Defense Office of the Inspector General, 2004). Both of these systems are only capable  
77 of operating through Sea State 2 (SS2, refer to Fort Eustis Weather (2012) for definitions of sea  
78 states) conditions, and improvements were made upon them for future causeway solutions (Garala,  
79 2004; Fort Eustis Weather, 2012).

80 The Navy Elevated Causeway System (ELCAS) and the Navy Modular Elevated Causeway  
81 System (ELCAS (M)) were designed in 1975 and 1985 respectively. To this day, the ELCAS is  
82 one of the most efficient deployable causeway systems to transport supplies over the surf-line. It  
83 is made from modular sections which are elevated on piles 6.10 m above water (Groff, 1992).  
84 Construction of this system begins at the beach where the piles are driven into the ground and  
85 the modules are connected between them (Lin, 1999). Initially the modules float on the water,  
86 and one by one they are lifted up and welded together (Skaalen and Rausch, 1977). The modular  
87 version was designed to facilitate quicker deployment (Groff, 1992). The 6.40 m wide causeway  
88 and turntable at the end allow for two-way traffic over the system. The intent was that this system  
89 would be operable in SS3 conditions, but it can only be operated through SS2 (Deitchman, 1993).

90 In 1991 the Joint Modular Lighter System (JMLS) was designed to address the shortcomings  
91 in the NLS and the MCS. Specifically, there existed interest to operate during SS3 conditions, but  
92 the system was only safe to operate through SS2 (Garala, 2004). Similar to the MCS, the 12.19  
93 m long by 2.44 m wide by 2.44 m high JMLS modules can be connected both side-to-side and  
94 end-to-end and can be made as powered or non-powered configurations. Unfortunately, several  
95 deficiencies were found with this system. Cracking in the welds between modules was seen with  
96 SS2 and above, the side connector system was problematic, and the many obstructions on the deck  
97 proved hazardous to personnel (Garala, 2004). The JMLS was replaced by the Improved Navy  
98 Lighterage System (INLS) that was designed in the 1990's. It is comprised of the same overall  
99 size and configuration as the JMLS, but incorporated improved side connectors that overcame the  
100 JMLS deficiencies. Operation is possible through SS3 conditions, the system is able to sustain  
101 only minimal damage under SS4, and it can structurally survive a SS5 event (Garala, 2004).

102 Despite these successes enjoyed by the MCS and INLS systems, both require strategic sealift  
103 assets to transport and deploy. This is a major drawback for rapid response operations by high  
104 speed shallow-draft vessels that could significantly shorten main supply routes by use of austere  
105 points of entry (such as damaged or small ports) that are much closer to the final destination. This  
106 is the main factor that led to the development of the LMCS. The reader is referred to Russell and  
107 Thrall (2013) for a review of portable and rapidly deployable bridges that provides additional detail  
108 on each of these systems with supporting images.

### 109 **Existing LMCS Prototype**

110 In order to address the shortcomings of existing causeway solutions for emergency response  
111 using shallow-draft vessels, a rapidly deployable floating causeway known as the LMCS was de-  
112 veloped (Figure 1). The existing prototype is comprised of 3.05 m long by 6.10 m wide modular  
113 units having an aluminum superstructure and pneumatic floats. The modules can be connected  
114 end-to-end to form a floating bridge or a floating causeway. High strength, but lightweight fabric  
115 is used for the floats to avoid puncture and abrasion. To take greater advantage of the buoyancy of  
116 the floats, the system must offer some flexibility. In the existing design, this flexibility is achieved  
117 by compliant connections - with a specific rotational compliance to provide this flexibility while  
118 meeting deflection limits - between the modules (Deming, 2009).

119 Based on the demand indicated by the Department of Defense and the shortcomings of exist-  
120 ing solutions, design priorities for the LMCS included (1) the ability to be transported aboard and  
121 deployed from shallow-draft vessels for use in austere environments (such as mudflats and wet-  
122 lands) and damaged ports or harbors (with shallow water or soft soil conditions), (2) the ability  
123 to be transported by land, air, and in an ISO compatible configuration, (3) easy assembly without  
124 substantial support equipment, and (4) low self-weight and packaged volume (Fowler et al., 2006;  
125 Deming, 2009). With these priorities in mind, the LMCS was designed to have minimal draft when  
126 unladen and be capable of increasing shoreline accessibility in shallow instances by partially de-  
127 flating floats on the leading edge as it is pushed or winched into position. For easy transportation,  
128 the LMCS was designed to be shipped in an ISO compatible configuration and transportable by

129 land, sea, or air. A key advantage in this respect, is that the system, unlike its predecessors, can be  
130 transported by the Joint High Speed Vessel (Fowler et al., 2006). Incorporation of these priorities  
131 will allow deployable causeways to access significantly shallower ports than was possible with  
132 previous systems. A 36.6 m causeway can be deployed by 7 people in 3 hours. The main actions  
133 required to deploy the structure are to inflate the floats by one of several methods and to join the  
134 modules together and activate the pinning mechanisms. The system can be retrieved in a compa-  
135 rable amount of time. The causeway is capable of supporting multiple M1A2 Main Battle Tanks  
136 when sufficient distance is allowed between them. The LMCS currently is 36.6 m comprised of 12  
137 modules that each provide 3.05 m of traveled way. In addition to being used in a Vessel to Shore  
138 Bridging mode, it has also been used successfully in a wet gap crossing application. During April  
139 2010, the LMCS was used in a post earthquake response exercise in connection with Exercise Arc-  
140 tic Edge 2010 in Alaska. This drill was completed by only 20 soldiers using organic equipment to  
141 offload and emplace 21.3 m of LMCS to provide an expedient floating bridge on the Eagle River.  
142 In order to secure the bridge against the current after deployment, it was anchored to land with  
143 several mooring lines. Other simulations were performed to demonstrate the utility of the system  
144 at an austere landing site and delivery via helicopter (Ferguson, 2010).

## 145 **RE-CONCEPTUALIZATION**

146       Though this system shows great promise and has performed well in recent field experiments  
147 and assessments, areas for further improvement include eliminating the compliant connections.  
148 These connections are complicated in design and are heavy, making up approximately half of  
149 the self-weight of each module. The goal of this research, therefore, was to re-conceptualize  
150 the existing design to eliminate these compliant connections. More specifically, the rotational  
151 compliance of the existing hinges was selected to permit sufficient rotation to take advantage of the  
152 buoyancy of the floats. Figure 2 represents three different configurations for a floating causeway,  
153 where the curved line indicates the water level. If the system is stiff, meaning that full moment  
154 connections exist between modules, then the structure cannot take full advantage of buoyancy  
155 between wave peaks (Figure 2A). The existing prototype, instead, employed compliant connections

156 between modules to better utilize this effect (Figure 2B, with a closeup of this connection shown  
157 in Figure 3A) (Resio et al., 2012). Since the connections are both heavy and complex, we aimed to  
158 eliminate the connections by transferring the rotational compliance to the deck itself (Figure 2C).  
159 This can be achieved by reducing the stiffness of the deck to a target value to achieve a specific  
160 rotation under a given load. Here, the target was for .08 radians of rotation over a 3.05 m long  
161 module under 1,140 kNm load (representing the design bending moment when the causeway is  
162 supporting one M1A2 tank and the ramp load) (Resio et al., 2012). Modules can then be connected  
163 using lighter, simpler fixed connections as proposed in Figure 3B. For transportation, modules  
164 could be disconnected.

165 In addition to eliminating the compliant connections, we also changed the cross-section to a  
166 box girder and investigated the use of advanced composite material and steel. Advanced composite  
167 materials were considered since they are lightweight and corrosion resistant, which is especially  
168 critical in salt-water environments that causeways experience. GFRP has been widely used in  
169 structural engineering projects and, since it is significantly less expensive than other advanced  
170 composites such as carbon fiber-reinforced polymers (CFRP), it was chosen for this research. Two  
171 versions of high strength steel were also considered as viable options for the re-conceptualization.

## 172 **STRUCTURAL OPTIMIZATION OF RE-CONCEPTUALIZED CAUSEWAY**

173 The design of deployable structures is a particularly challenging problem since self-weight and  
174 packaged volume are at a premium to permit transportability, and ultimately feasibility. Structural  
175 optimization is a useful tool to fully explore the design space. For the re-conceptualized system,  
176 we have implemented multi-objective structural optimization to minimize the self-weight and the  
177 stiffness, subject to the constraints of a minimum target value for the stiffness, structural criteria  
178 based on the Pre-Standard for Load & Resistance Factor Design (LRFD) of Pultruded Fiber Re-  
179 inforced Polymer (FRP) Structures manual by the American Society of Civil Engineers (LRFD  
180 manual, hereafter) (ASCE, 2010), and geometric requirements. This section will first review rele-  
181 vant research in this area and then present the problem formulation and algorithm employed.

182 Structural optimization has been used in the past to more efficiently design floating decks.

183 One example of this is a study to determine the optimal layout of gill cells in very large floating  
184 structures (VLFS) to minimize the differential deflection of the system due to non-uniform loading  
185 (Wang et al., 2007; Pham and Wang, 2010). The introduction of specific cells which allow water  
186 to flow freely in and out can reduce this differential deflection by up to 66% (Pham and Wang,  
187 2010). Studies have been made to determine the optimal number and location of these cells for  
188 various shaped VLFS. Circular structures produce a differentiable and continuous optimization  
189 function. Therefore, sequential quadratic programming, a classical optimization algorithm based  
190 on Newton's method, was used (Wang et al., 2007). When optimizing the arrangement of the gill  
191 cells in an arbitrarily shaped section with any loading configuration, the genetic algorithm was  
192 utilized (Pham and Wang, 2010).

193 Optimization strategies have also been used in the design of permanent bridge decking systems.  
194 Many of these have been in response to the deterioration of existing bridges. The most widely used  
195 optimization algorithm for this type of analysis is the Genetic Algorithm (GA) because of its capa-  
196 bilities to escape local minima and to handle discrete and non-differentiable optimization functions.  
197 GA and Shuffled Frog Leaping were utilized to determine which bridge decks in a region should be  
198 repaired first (Elbehairy et al., 2006). Multi-objective GA has also been used to determine bridge  
199 deck rehabilitation so that the total rehabilitation cost and the weighted average deterioration de-  
200 gree were minimized. With the many solutions given by multi-objective optimization, a bridge  
201 owner would then be able to select the best combination for his or her particular case (Liu et al.,  
202 1997). As mentioned previously, fiberglass reinforced polymers (FRP) are also becoming popular  
203 in bridge deck rehabilitation projects. However, a substantial up-front cost is typically required for  
204 these components. While the examples given above aim to reduce the life-time cost of the bridge  
205 system, the optimization examples given below are geared at reducing this initial cost so that FRP  
206 is a more appealing option to bridge owners. To achieve this goal, the volume of the decking sys-  
207 tem is minimized. This has been done on its own (Dey et al., 2013), and also in conjunction with  
208 optimization of the FRP material composition (Park et al., 2005; He and Aref, 2003). GA was used  
209 for all three of these studies. Finally, in order to account for uncertainties such as those that exist



210 in the material properties, structural dimensions, and applied loads, another study was performed  
 211 to simultaneously consider a finite element model, the optimization algorithm, and a reliability  
 212 analysis procedure (Thompson et al., 2006).

### 213 **Problem Formulation**

214 For the proposed re-conceptualization, the authors sought a low-weight design with a target  
 215 stiffness. To achieve this goal, multi-objective structural optimization for minimum self-weight  
 216 and minimum moment of inertia (related to stiffness by young's modulus) was implemented, with  
 217 a constraint that the moment of inertia not be below the target value. Additional constraints include  
 218 that the design meet the structural requirements of the LRFD manual and geometric criteria related  
 219 to function and packaging. The design variables relate to the box girder cross-section, including  
 220 the depth of the entire cross-section ( $H$ ), the thickness of the top and bottom flanges ( $t_{ft}$  and  $t_{fb}$ ,  
 221 respectively), the width of the bottom flange ( $w_{fb}$ ), and the thickness of the exterior webs ( $t_{we}$ )  
 222 (Figure 4). These variables are permitted to range from 9.5 to 6,096 mm in 3.2 mm increments.  
 223 The minimum thickness and discrete increment size is based on the manufacturing capabilities for  
 224 FRP members. The number ( $n$ ) and thickness of the interior webs ( $t_{wi}$ ) are determined within  
 225 the algorithm to ensure compliance with all code requirements. Additionally shown in Figure 4  
 226 are the width of the top flange ( $w_{ft}$ ) which is a fixed value of 6,096 mm based on functionality  
 227 requirements, the height of the webs ( $h_w$ ), and the horizontal width of the exterior flange ( $b_{we}$ ).  
 228 These last two variables are geometrically related to the design variables in the following way:

$$h_w = H - t_{ft} - t_{fb} \quad (1)$$

229

$$b_{we} = \frac{t_{we}}{\sin\left(\arctan\frac{h_w}{0.5(w_{ft}-w_{fb})}\right)} \quad (2)$$

230 These are used to simplify equations presented here.

231 The formal definition of this optimization problem is as follows:

$$\begin{aligned}
 & \underset{H, t_{ft}, t_{fb}, w_{fb}, t_{we}}{\text{minimize}} & W(s) &= p(t_{ft}w_{ft} + t_{fb}w_{fb} + 2b_{we}h_w + n_{wi}t_{wi}h_w) \\
 & & I_x(s) &= \sum_{i=1}^{nel} (I_{xi} + A_i d_i^2) \\
 & \text{such that} & c_i &\leq 0; i = 1, \dots, u
 \end{aligned} \tag{3}$$

232 where  $W$  refers to the self-weight of the superstructure, which is the summation of the area of each  
 233 of the of elements times the density ( $p$ ) of the GFRP material. The moment of inertia objective  
 234 function ( $I_x$ ) is simply found by the parallel axis theorem, where  $d$  is the distance from an individ-  
 235 ual component's centroid to that of the overall superstructure assembly's. Finally, the cross section  
 236 produced by the algorithm must conform to the  $u$  number of constraints,  $c$ , related to the minimum  
 237 value for the moment of inertia ( $c_1$ ), structural constraints of the LRFD manual ( $c_2$ - $c_6$ ), and geo-  
 238 metric constraints ( $c_7$ - $c_8$ ) related to design criteria for use and packaging. Prior three-dimensional  
 239 finite element analyses of the system, performed by the ERDC, provide the design moment, shear,  
 240 and torsion acting on the system due to load combinations prescribed by Trilateral Design code  
 241 which include the effects of dead, vehicle, wave, ramp, and damaged pontoon loads (Resio et al.,  
 242 2012; Federal Republic of Germany, United Kingdom & United States of America, 1996). Table  
 243 1 provides these values. A load factor of 1.33, which was the largest load factor provided in the  
 244 Trilateral Design code, was then conservatively applied to each to enable design by LRFD (Federal  
 245 Republic of Germany, United Kingdom & United States of America, 1996). These values would  
 246 need to be adjusted for the final detailed design, but they provide sufficient detail for this stage of  
 247 the design process. The LRFD manual was used for minimum values for material properties. If  
 248 these values were not available, then the appropriate values were taken from the design manual  
 249 Fiberglass Grating and Structural Products by Delta Composites (Delta Composites L.L.C., 2004).

250 The first constraint,  $c_1$ , ensures that the superstructure moment of inertia does not allow greater  
 251 flexibility of the system than desired by setting a lower limit on this value, as follows:

$$c_1 = M_s \frac{\rho}{E_L} - I_x \leq 0 \quad (4)$$

252 where  $\rho$  is the desired radius of curvature,  $M_s$  is the bending moment under service load (Table 1),  
 253 and  $E_L$  is the longitudinal modulus of elasticity. In using this moment-curvature relationship, we  
 254 are assuming that plane sections remain plane and linear elastic material behavior. Given a desired  
 255 rotation of .08 radians over a 3.05 m long module under a uniform bending moment, the target  
 256 (minimum) moment of inertia is 0.003503 m<sup>4</sup>.

257 Structural constraints were defined by the criteria in the LRFD manual, related to shear (strength  
 258 of members due to material rupture in shear, to web shear buckling, and to web lateral stability),  
 259 flexure (strength of members due to material rupture and due to local instability), torsion (torsional  
 260 capacity when strength governs), and concentrated load requirements (strength of members due to  
 261 tensile rupture in the webs, to web crippling, to web compression buckling, and to flange flexural  
 262 failure). Lateral torsional buckling and torsional effects related to warping and bending have not  
 263 been included since these effects are generally negligible for closed box sections. The following  
 264 paragraphs will detail each of these structural constraints. For all the calculations it was assumed  
 265 that all GFRP components have the same material properties and that no delamination or separation  
 266 occurs between them.

267 The LRFD design for members in shear is based on the governing behavior between material  
 268 rupture in shear and web shear buckling. The shear capacity ( $V_n$ ) must exceed the demand ( $V_u$ ):

$$c_2 = V_u - \lambda\phi V_n \leq 0 \quad (5)$$

269 where  $\lambda$  is a time effect factor and  $\phi$  is the resistance factor for shear. For this calculation as well as  
 270 for all others presented in this document,  $\lambda = 0.8$  (ASCE, 2010). The shear demand on the system  
 271 is the design value identified in Table 1 multiplied by the 1.33 load factor. For material rupture  
 272 in shear,  $\phi = 0.65$  and  $V_n$  is calculated as  $V_n = F_{LT} A_s$ , where  $F_{LT}$  is the characteristic in plane  
 273 shear strength, and  $A_s$  is the shear area ( $A_s = Hb_{we}$ ). This shear area was conservatively assumed

274 to be the area of a single web. In reality the shear would be transferred through the section by  
 275 several, or all, of the webs. However, since the exact distribution of the contribution from each  
 276 web could only be known by a detailed finite element analysis, the most conservative assumption  
 277 was made that any individual web must be able to carry the entire demand. For the final design  
 278 the web widths could be reduced after a detailed analysis was performed to determine the actual  
 279 demand on each web. For web shear buckling,  $\phi = 0.80$  and  $V_n$  is found by  $V_n = F_{cr}A_s$ , where  
 280  $f_{cr}$  is the critical shear buckling stress, a function of the design variables  $t_{we}$  and  $H$ . The reader is  
 281 referred to the LRFD manual for this equation (ASCE, 2010). Note that these constraints consider  
 282 the adequacy of the external webs only. The design of the internal webs are discussed at the end of  
 283 this section of the paper.

284 The next two constraints check the adequacy of the section against the flexural demands. The  
 285 constraint  $c_3$  ensures that the material will not rupture in bending. The bending capacity ( $M_n$ ) must  
 286 exceed the demand ( $M_u$ ) as follows:

$$c_3 = M_u - \lambda\phi M_n \leq 0 \quad (6)$$

287 where the resistance factor,  $\phi$ , is 0.65. The nominal flexural strength, is calculated as  $M_n = \frac{F_L I_x}{y}$ ,  
 288 where  $F_L$  is the longitudinal flexural strength and  $y$  is the distance from the neutral axis of the  
 289 cross section to the extreme fiber of the member. Here  $M_u$  is the factored moment, defined as 1.33  
 290 times the design moment in Table 1. The second flexural constraint checks that the external webs  
 291 are thick enough to prevent local instability. The constraint is defined as:

$$c_4 = t_{we\ r} - t_{we} \leq 0 \quad (7)$$

292 where  $t_{we\ r}$  represents a required web thickness that can be found by rearranging the equations  
 293 from section 5.2.3 of the LRFD manual as follows:

$$t_{we\ r} = \sqrt{\frac{6H^2}{11.1\pi^2(1.25\sqrt{E_{L,w}E_{T,w}} + E_{T,w}\nu_{LT} + 2G_{LT})} \frac{M_u y}{\lambda\phi I_x}} \quad (8)$$

294 where  $\phi = 0.80$ ,  $E_{L,w}$  and  $E_{T,w}$  are respectively the longitudinal and transverse modulus of elasticity  
 295 in the web,  $\nu_{LT}$  is the longitudinal poisson's ratio, and  $G_{LT}$  is the in plane shear modulus. The  
 296 flanges and internal webs were also checked against local instabilities and are discussed in further  
 297 detail later.

298 The fifth constraint,  $c_5$ , relates to the torsional capacity of the member, and is defined as

$$c_5 = T_u - \lambda\phi T_n \leq 0 \quad (9)$$

299 where  $T_u$  is the required torsional demand (1.33 times the design value in Table 1),  $\phi$  is 0.70, and  
 300  $T_n$  is the nominal torsional capacity. The torsional capacity for a closed section can be calculated  
 301 as  $T_n = 2tF_{LT}A_o$ , where  $t$  is conservatively taken to be the minimum thickness of any of the  
 302 exterior elements and  $A_o$  is the area enclosed by the centerline of the exterior elements (Beer et al.,  
 303 2006).

304 For concentrated loads, the load demand ( $R_u$ ) must exceed the capacity ( $R_n$ ) as follows:

$$c_6 = R_u - \lambda\phi R_n \leq 0 \quad (10)$$

305 The demand arises from concentrated wheel loads from M1A2 Abrams Main Battle Tank and from  
 306 the ramp which connects the ship to the causeway. The ramp load was considered to be a special  
 307 loading scenario which the entire causeway system need not be able to withstand. It was assumed  
 308 that special detailing would be considered in the module which would withstand the ramp load.  
 309 Therefore, the concentrated load was taken to be the weight of the M1A2 tank divided by the  
 310 number of wheels ( $R_u=68\text{kN}$ , including the load factor). Three limit states (tensile rupture, web  
 311 crippling, and compressive buckling) are included in this constraint. For all three of these checks,  
 312 both the internal and the external webs were analyzed, and the smallest  $\phi R_n$  value from the three  
 313 checks governed. For tensile rupture in the webs,  $R_n = l_{ten}F_{T,w}t_w$ , where  $l_{ten}$  is the depth of the  
 314 webs, and  $F_{T,w}$  is the transverse flexural strength of the webs. It was again conservatively assumed  
 315 that a single web was required to carry the entire load. Therefore,  $t_w$  was taken as the width of

316 the thinnest web (either  $t_{we}$  or  $t_{wi}$ ). For this limit state,  $\phi$  is 0.65. For web crippling,  $\phi = 0.70$  and  
317  $R_n = 0.7h_w t_w F_{sh,int} (1 + \frac{2k+6t_{plate}+b_{plate}}{d_w})$ , where  $F_{sh,int}$  is the interlaminar shear strength and  $k$  is  
318 defined as the distance from the top of the member to the bottom of the fillet. Since no fillet exists  
319 in the cross-section this was taken as the thickness of the top flange,  $t_{ft}$ .  $t_{plate}$  and  $b_{plate}$  refer to  
320 the thickness and length of the bearing plate. Since no bearing plate exists, this again was taken  
321 as the thickness of the top flange and the maximum allowable value of 102 mm, respectively.  $d_w$   
322 is the depth of the web, which is  $h_w$  in this case. For web compression buckling,  $\phi = 0.80$  and  
323  $R_n = f_{cr} A_{eff}$ , where  $A_{eff}$  is the effective area ( $A_{eff} = l_{eff} t_w$ , where  $l_{eff}$  is the lesser of the web  
324 depth ( $d_w$ ) or the distance between vertical stiffeners) and  $f_{cr}$  is

$$f_{cr} = \frac{\pi^2 t_w^2}{6l_{eff}^2} (\sqrt{E_{L,w} E_{T,w}} + E_{T,w} \nu_{LT} + 2G_{LT}) \quad (11)$$

325 One more concentrated load check appears in the LRFD manual. This last equation is associated  
326 with the stability of the flange and is discussed later.

327 The remaining constraints are geometric, relating to functional needs, packaging requirements,  
328 and physical constraints. The angle between the top and bottom flange must be greater than 45%,  
329 which can be expressed formally as:

$$c_7 = w_{ft} - w_{fb} - 2H \leq 0 \quad (12)$$

330 For packaging, the depth of the cross section must be shallow enough to allow four modules to fit  
331 in the footprint of one ISO container (which corresponds to a maximum depth of 400mm):

$$c_8 = H - 400\text{mm} \leq 0 \quad (13)$$

332 The final two geometric constraints ensure that the algorithm does not specify a cross section that  
333 is physically impossible. Specifically, the thickness of the top and bottom flanges cannot be greater  
334 than the depth of the entire member:

$$c_9 = t_{ft} + t_{fb} - H \leq 0 \quad (14)$$

335 and the thickness of the external webs cannot be greater than the width of the cross section:

$$c_{10} = 2t_{we} - w_{ft} \leq 0 \quad (15)$$

336 The number and thickness of the internal webs have not been set as design variables, but instead  
 337 have been calculated to meet various limit states. The required number of internal webs is based  
 338 on meeting stability criteria for the flanges. This includes the strength of the compression flange  
 339 member due to local instability induced by flexure on the entire cross section and against flange  
 340 flexural failure caused by a concentrated force. The equation to determine the required number  
 341 of internal webs ( $n_{wi}$ ) based on the former of these is too complex to include here. It is based  
 342 on Sections 5.2.3 and 5.2.3.4, part a of the LRFD manual, “Compression flange local buckling  
 343 for square and rectangular box members”. Since both the top and the bottom flange of the cross  
 344 section can go into compression it was necessary to perform this analysis on each of them. The  
 345 calculation for  $n_{wi}$  based on concentrated load demands is:

$$n_{wi} = \frac{3W_{ft}R_u}{\lambda\phi F_{T,f}t_{ft}^2} - 1 \quad (16)$$

346 where  $F_{T,f}$  is the transverse flexural strength in flange, and  $\phi = 0.65$ .  $n_{wi}$  is taken as the governing  
 347 value between these two calculations. It was assumed that the internal webs were spaced evenly  
 348 across the section and that the concentrated force acted in the most critical location, that is midway  
 349 between two webs. The web thickness was determined by the governing calculation between  
 350 the material rupture in shear, web shear buckling, and the flexural local instability. These first  
 351 two checks are derived from the same equations as described in  $c_2$ ; the equations were simply  
 352 rearranged to solve for  $t_{wi}$  (which replaces  $b_{we}$  and  $t_{we}$  in the  $c_2$  equations). Similarly, the required  
 353 thickness of the internal webs based on the flexural instability check is identical to that described  
 354 in  $c_4$ . As was the case with the external web calculations, it was conservatively assumed that each

355 individual web would be required to carry the entire shear demand.

356 As noted earlier, we performed an analogous procedure for two types of high strength steel -  
357 A709 HPS 70 steel (483MPa yield strength) and A709 HPS 100 steel (689 MPa yield strength).  
358 The design variables and objective functions remain the same, but the structural constraints ( $c_2$ - $c_6$ )  
359 are replaced by the constraints from the American Association of State Highway and Transporta-  
360 tion Officials (AASHTO) LRFD Bridge Design Specifications (AASHTO, 2012) for the design of  
361 steel box girders. The target moment of inertia ( $c_1$ ) was also changed to achieve the same desired  
362 stiffness with a different modulus of elasticity ( $0.000217 \text{ m}^4$ ).

### 363 **Simulated Annealing**

364 The authors chose to approach their problem using Multi-Objective Simulated Annealing (MOSA).  
365 Heuristic algorithms, such as Simulated Annealing (SA), are intuitive, relatively quick, and are ca-  
366 pable of handling the continuous and discrete constraints in current structural design codes. For the  
367 specific applications of bridge engineering, simulated annealing has been used to design reinforced  
368 concrete box road structures (Perea et al., 2008), bridge piers (Martinez-Martin et al., 2012), pre-  
369 stressed concrete and tensegrity footbridges (Martí and González-Vidosá, 2010; Ali et al., 2010),  
370 for shape and sizing optimization of linkage-based movable bridges (Thrall et al., 2012), and to  
371 identify moving axle loads to aid in design (in conjunction with Genetic Algorithms) (Qu et al.,  
372 2011). Note that this is a non-exhaustive review of the implementation of SA for bridge design, but  
373 serves to highlight potential applications of this algorithm in this field. The authors have already  
374 demonstrated the effectiveness of SA for the design of deployable structures comprised of linkages  
375 (Thrall et al., 2013; Thrall, 2011).

376 SA is an iterative improvement algorithm that is based on an analogy to crystal formation. In  
377 the physical process of annealing, as the melted mass is slowly cooled, the energy of the system  
378 gradually decreases. During this process there is a probability ( $P$ ) that a higher energy configura-  
379 tion can occur, which ultimately leads to a lower energy configuration. This probability is given  
380 by the formula,  $P = e^{\frac{-\Delta E}{T}}$  where  $\Delta E$  refers to the difference in the energy configurations and  $T$   
381 is the temperature of the mass. As the temperature is decreased there is a lower probability that a



382 higher energy state will occur (Kirkpatrick et al., 1983). This process can be extended to structural  
383 optimization. This was first proposed by Kirkpatrick et al. (1983), where the probability relates  
384 to the probability of accepting higher value functions and energy relates to the existing function,  
385 therefore enabling the algorithm to escape local minima.  $T$  is a variable that can be controlled by  
386 the user. A high temperature is used initially to widely explore the solution space and is slowly  
387 decreased as the algorithm converges (Kirkpatrick et al., 1983).

388 For single objective SA (e.g minimum weight optimization), the algorithm begins by selecting  
389 an initial solution by randomly generating a feasible set of design variables from a database of  
390 discrete values specified by the user. This set becomes the initial best solution. One or more of  
391 the variables are then randomly perturbed to generate a new solution. If this solution conforms to  
392 all constraints and produces a lower weight solution, it becomes the new current solution. If not,  
393 there is a certain probability, as discussed above, that it can still be accepted as the current solution  
394 upon which the algorithm continues to iterate. The algorithm explores the solution space for a user  
395 defined number of iterations in a cooling cycle. The temperature is decreased at the end of each  
396 cooling cycle, thereby decreasing the probability that a higher weight solution will be accepted. In  
397 other words, the algorithm's ability to escape a local minimum decreases. Convergence is defined  
398 as a certain number of cooling cycles in which there has been no improvement in the solution. The  
399 final result is the lowest weight solution (Kirkpatrick et al., 1983).

400 This process of optimizing a single design variable can be extended to multi-objective optimiza-  
401 tion. Rather than combining objective functions using a weighted average, the objective functions  
402 remained separated to produce a pareto-optimal set of solutions. Solutions are pareto-optimal if  
403 they are not overshadowed by other solutions in either objective function. At convergence a pareto-  
404 optimal set gives the designer an array of possible solutions spanning between extremes that one  
405 would find through single objective optimization. Based on the design priorities, a designer can  
406 select a final solution (Suppaitnarm et al., 2000).

## 407 **Optimized Designs**

408 By nature, heuristic algorithms are not guaranteed to converge on the same solution each time  
409 they are employed. The user defines several parameters, including  $v$  (the maximum number of  
410 variables to be varied at once),  $pm$  (the amount of perturbation permitted along the database of  
411 allowable values),  $r$  (the factor by which the temperature is reduced),  $m$  (the length of a cooling  
412 cycle), and  $n$  (the number of cooling cycles for convergence of the algorithm). The quality and  
413 robustness of the algorithm are dependent on the selection of these parameters. To determine a  
414 robust selection of these parameters for multi-objective optimization, single objective optimization  
415 was performed for each objective function using 16 different combinations of the parameters,  $v$ ,  
416  $pm$ ,  $r$ ,  $m$ , and  $n$ . Twenty numerical simulations were performed for each combination. Table 2  
417 provides the combinations considered as well as the average result ( $\mu$ ), standard deviation ( $\sigma$ ), and  
418 coefficient of variation for the GFRP design ( $c_v$ ). The most robust combination (where robustness  
419 is defined as having both a low average and a low standard deviation) for both objective functions  
420 is SA 4 ( $v = 1$ ,  $pm = 10$ ,  $r = 0.8$ ,  $m = 10,000$ , and  $n = 2$ , highlighted in bold in Table 2).

421 With this selection of parameters, MOSA was performed for the GFRP design to find a pareto-  
422 optimal set of results from which a designer can select a final design (Figure 5, empty circles). The  
423 diamond and square show the best results from single objective optimization of the weight and  
424 moment of inertia, respectively, representing the extremes between which the pareto-optimal set  
425 spans. The dotted horizontal line shows the target moment of inertia value. The authors chose the  
426 cross section shown as a black filled circle as the best combination of moment of inertia and weight.  
427 This solution had a moment of inertia close to the target value ( $0.003504 \text{ m}^4$ ), with an acceptable  
428 weight ( $1,208 \text{ kg/m}$ , including the weight of the floats). The corresponding cross section is shown  
429 in Figure 6A. An analogous procedure was performed for the two grades of steel and the resulting  
430 optimized cross sections are shown in Figure 6B,C.

431 The optimized results are summarized and compared against the existing LMCS system in  
432 Table 3. For all three optimized designs, the desired system stiffness was achieved (or closely  
433 approached). Therefore, the compliant connections can be eliminated as desired and replaced by

434 the simplified option shown in Figure 3B. In addition to simplifying the connection detail, one  
435 of the highest priorities was reducing the total system weight. Table 3 provides the weight of  
436 the modules alone (W, without hinges or floats), the weight of the modules with connections (no  
437 floats), and the total system weight which includes the floats. Note that it was assumed that the  
438 simplified connections proposed for the re-conceptualized system would have a negligible impact  
439 on the weight of the entire system. Unfortunately, the GFRP design was not able to reduce the  
440 system weight below that of the existing prototype. This is largely due to local constraints on  
441 the top flange. Both steel designs showed lower weights than the GFRP option, with the 689  
442 MPa grade option having a lower system weight than the existing system. Furthermore, the total  
443 superstructure depth (H) is reduced, thereby enabling additional modules to fit within the footprint  
444 of an ISO container. This offers the potential to reduce the time and cost for transportation to the  
445 site. Overall, these results are very promising.

## 446 **CONCLUSIONS AND FUTURE WORK**

447 The authors have presented a re-conceptualized, rapidly deployable causeway with the use  
448 of multi-objective structural optimization for GFRP and two grades of high strength steel. The  
449 resulting designs (Figure 6, Table 3) are highly promising, with each meeting (or approaching) the  
450 desired target stiffness to replace the heavy and complex compliant connections (Figure 3A) with  
451 a proposed simplified connection (Figure 3B). It is expected that this proposed connection will be  
452 simpler, lighter, and easier to deploy and maintain than the compliant connections of the original  
453 prototype. This will result in a simpler superstructure design for fabrication and deployment.  
454 Each optimized design also reduces the cross-section depth, thereby reducing the cost and time for  
455 transportation. While the GFRP and 483 MPa steel designs were not able to reduce the overall  
456 system weight, the 689 MPa steel design does, thereby achieving a design priority of the project.

457 The results from the authors' study of the re-conceptualized causeway have been presented  
458 here. This work is intended as an initial, optimized design upon which a final detailed design  
459 would need to be performed. Areas which warrant further study for such a final design include  
460 further detailed connection design. Additional finite element analysis could also be performed to

461 determine if it is possible to further reduce the weight by reducing the width of the webs, as the  
462 selected width was determined by conservatively assuming that each web would carry the entire  
463 shear demand. The cost for material and manufacturing should also be investigated. Finally, the  
464 constructability of the proposed cross sections and connection details between components need  
465 to be examined. While such future work is necessary to lead to a detailed design, this research has  
466 culminated in preliminary, optimized designs for a re-conceptualized, rapidly deployable cause-  
467 way.

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## 472 **REFERENCES**

473 AASHTO (2012). *AASHTO Load & Resistance Factor Design (LRFD) Specifications, Customary*  
474 *US Units*. 6th edition American Association of State Highway and Transportation Officials  
475 (AASHTO).

476 Ali, N. B. H., Rhode-Barbarigos, L., Albi, A. A. P., and Smith, I. F. C. (2010). “Design optimiza-  
477 tion and dynamic analysis of a tensegrity-based footbridge.” *Engineering Structures*, 32(11),  
478 3650–9.

479 Anon. (2012). “Modular causeway systems. Global Security Website.  
480 <<http://www.globalsecurity.org/military/systems/ship/mcs.htm>> (April 21, 2012).

481 ASCE (2010). *Pre-standard for Load & Resistance Factor Design (LRFD) of Pultruded Fiber*  
482 *Reinforced Polymer (FRP) Structures (Final)*. American Society of Civil Engineers (ASCE)  
483 American Composites Manufacturers Associations (ACMA).

484 Beer, F. P., Johnston, E. R., and DeWolf, J. T. (2006). *Mechanics of Materials*. McGraw Hill  
485 Higher Education, Boston, 4th edition.

486 Buonopane, M. (2002). “Modular Causeway Systems: Hitting the beach with the U.S. Army.”  
487 *Proceedings of the Seventh International Conference on Applications of Advanced Technology*  
488 *in Transportation*, ASCE, Cambridge, MA, 241–248.

489 Deitchman, C. G. (1993). *Possible Logistical Implications of ‘From the Sea’*. Naval War College,  
490 Newport, RI (June).

491 Delta Composites L.L.C. (2004). “Design manual: fiberglass grating and structural products.  
492 <[http://www.deltacomposites.com/lit\\_library/DelDesMan.pdf](http://www.deltacomposites.com/lit_library/DelDesMan.pdf)> (January 20, 2013).

493 Deming, M. A. (2009). “Lightweight Modular Causeway System: Logistics advanced concept  
494 technology demonstration.” *Army Logistician*, Professional Bulletin of United States Army Lo-  
495 gistics, 50–51.

496 Department of Defense Office of the Inspector General (2004). *Contract Award and Adminis-*  
497 *tration for Modular Causeway Systems (D-2005-021)*. Department of Defense, Arlington, VA  
498 (November).

499 Dey, T. K., Srivastava, I., Khandelwal, R. P., Sharma, U. K., and Chakrabarti, A. (2013). “Optimum  
500 design of FRP rib core bridge deck.” *Composites Part B: Engineering*, 45(1), 930–938.

501 Elbehairy, H., Elbeltagi, E., Hegazy, T., and Soudki, K. (2006). “Comparison of two evolutionary  
502 algorithms for optimization of bridge deck repairs.” *Computer-Aided Civil and Infrastructure*  
503 *Engineering*, 21, 561–572.

504 Federal Republic of Germany, United Kingdom & United States of America (1996). *Trilateral*  
505 *design and test code for military bridging and gap-crossing equipment*. United States.

506 Ferguson, B. (2010). “State of the art equipment bridges the gap. AMM-  
507 TIAC: Advanced Materials, Manufacturing and Testing Information Analysis Center,  
508 <<http://www.af.mil/news/story.asp?id=123202740>> (December 2, 2011).

509 Fort Eustis Weather (2012). “Pierson - Moskowitz sea spectrum.  
510 <[http://www.eustis.army.mil/WEATHER/Weather\\_Products/seastate.htm](http://www.eustis.army.mil/WEATHER/Weather_Products/seastate.htm)> (April 26, 2012).

511 Fowler, J. E., Resio, D. T., Pratt, J. N., Boc, S. J., and Sargent, F. E. (2006). “Innovations for future  
512 gap crossing operations.” Engineering Research and Development Center, Vicksburg, MI, 1–5.

513 Garala, H. J. (2004). “Development of a composite prototype module for the Improved Navy  
514 Lighterage System (INLS).” *Proceedings of the Fourteenth International Offshore and Polar  
515 Engineering Conference*, International Society of Offshore and Polar Engineers, Toulon, France,  
516 235–243.

517 Groff, H. L. (1992). “Overview and analysis of the U.S. Navy Elevated Causeway System. MSE  
518 Thesis, University of Texas at Austin, Austin, TX.

519 He, Y. and Aref, A. J. (2003). “An optimization design procedure for fiber reinforced polymer  
520 web-core sandwich bridge deck systems.” *Composite Structures*, 60, 183–195.

521 Kirkpatrick, S., Gelatt, C. D., and Vecchi, M. P. (1983). “Optimization by simulated annealing.”  
522 *Science*, 220(4598), 671–680.

523 Lin, S. S. (1999). “Development of a rapid pile splicer for the Navy Modular Elevated Causway  
524 System.” *Proceedings of the Ninth International Offshore and Polar Engineering Conference*,  
525 International Society of Offshore and Polar Engineers, Brest, France, 554–557.

526 Liu, C., Hammad, A., and Itoh, Y. (1997). “Multiobjective optimization of bridge deck rehabilita-  
527 tion using a genetic algorithm.” *Microcomputers in Civil Engineering*, 12, 431–443.

528 Martí, J. V. and González-Vidoso, F. (2010). “Design of prestressed concrete precast pedestrian  
529 bridges by heuristic optimization.” *Advances in Engineering Software*, 41, 916–922.

530 Martinez-Martin, F. J., Gonzalez-Vidoso, F., Hospitaler, A., and Yepes, V. (2012). “Multi-objective  
531 optimization design of bridge piers with hybrid heuristic algorithms.” *Journal of Zhejiang Uni-  
532 versity - Science A*, 13(6), 420–432.

533 Park, K. T., Kim, S. H., Lee, Y. H., and Hwang, Y. K. (2005). "Pilot test on a developed GFRP  
534 bridge deck." *Composite Structures*, 70, 48–59.

535 Perea, C., Alcalá, J., Yepes, V., Gonzalez-Vidoso, F., and Hospitaler, A. (2008). "Design of rein-  
536 forced concrete bridge frames by heuristic optimization." *Advances in Engineering Software*, 39,  
537 676–688.

538 Pham, D. C. and Wang, C. M. (2010). "Optimal layout of gill cells for very large floating struc-  
539 tures." *Journal of Structural Engineering*, ASCE, 907–916.

540 Potts, K. (2009). "Construction during World War II: Management and financial administration."  
541 *Proceedings of the 25th Annual ARCOM Conference*, Association of Researchers in Construc-  
542 tion Management, Nottingham, UK, 847–856 (September).

543 Qu, W., Wang, Y., and Pi, Y. (2011). "Multi-axle moving train loads identification on simply sup-  
544 ported bridge by using simulated annealing genetic algorithm." *International Journal of Struc-  
545 tural Stability and Dynamics*, 11(1), 57–71.

546 Resio, D. T., Fowler, J. E., Boc, S. J., Padula, J. A., and Holder, P. M. (2012). "Development  
547 and testing of the lightweight modular causeway system. Manuscript in preparation, U.S. Army  
548 Engineer Research and Development Center, Vicksburg, MS.

549 Russell, B. R. and Thrall, A. P. (2013). "Portable and rapidly deployable bridges: Historical  
550 perspective and recent technology developments." *Journal of Bridge Engineering*, ASCE.

551 Skaalen, C. I. and Rausch, A. B. (1977). "Container off-loading and transfer system (COTS) -  
552 advanced development tests of elevated causeway system. Volume II - elevated causeway instal-  
553 lation and retrieval. Civil Engineering Laboratory, Port Hueneme, CA, 1-22, 31-33.

554 Suppaitnarm, A., Seffen, K. A., Parks, G. T., and Clarkson, P. J. (2000). "A simulated annealing  
555 algorithm for multiobjective optimization." *Engineering Optimization*, 33(1), 59–85.

- 556 Thompson, M. D., Eamon, C. D., and Rais-Rohani, M. (2006). “Reliability-based optimization  
557 of fiber-reinforced polymer composite bridge deck panels.” *Journal of Structural Engineering*,  
558 ASCE, 132(12), 1898–1906.
- 559 Thrall, A. P. (2011). “Shape-finding of a deployable structure using simulated annealing.” *Journal*  
560 *of the International Association for Shell and Spatial Structures*, 52(4), 241–247.
- 561 Thrall, A. P., Adriaenssens, S., Paya-Zaforteza, I., and Zoli, T. P. (2012). “Linkage-based movable  
562 bridges: Design methodology and three novel forms.” *Engineering Structures*, 37, 214–223.
- 563 Thrall, A. P., Zhu, M., Guest, J. K., Paya-Zaforteza, I., and Adriaenssens, S. (2013). “Structural  
564 optimization of deploying structures comprised of linkages.” *Journal of Computing in Civil En-*  
565 *gineering*, ASCE Accepted for publication.
- 566 Wang, C. M., Pham, D. C., and Ang, K. K. (2007). “Effectiveness and optimal design of gill cells  
567 in minimizing differential deflection in circular VLFS.” *Engineering Structures*, 29, 1845–1853.



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572 *the name of the combination and the parameters, the next 3 columns provide the*  
573 *results for the minimum weight objective function, and the final 3 columns provide*  
574 *the results for the minimum moment of inertia objective function. The bold row*  
575 *indicates the most robust solution.* . . . . . 27

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<i>Type</i>	<i>Value</i>
Moment	1140 kNm
Shear	360 kN
Torison	160 kNm

**TABLE 1. Design Moment, Shear, and Torsion. Data courtesy of US Army ERDC [Resio et al 2012]**

Parameters						Minimum Area			Minimum Ix		
Name	$v$	$pm$	$r$	$m$	$n$	$\mu (m^2)$ $\times 10^{-1}$	$\sigma (m^2)$ $\times 10^{-3}$	$c_v$	$\mu (m^4)$ $\times 10^{-3}$	$\sigma (m^4)$ $\times 10^{-9}$	$c_v$
SA 1	1	10	0.8	5	1	6.22	1.3	0.21	3.50	21	6.1
SA 2	1	10	0.8	10	2	6.22	0.61	0.10	3.50	7.7	2.2
SA 3	1	10	0.9	5	1	6.45	73	11.29	3.50	14	4.1
<b>SA 4</b>	<b>1</b>	<b>10</b>	<b>0.9</b>	<b>10</b>	<b>2</b>	<b>6.22</b>	<b>0.46</b>	<b>0.07</b>	<b>3.50</b>	<b>7.2</b>	<b>2.1</b>
SA 5	1	10	0.8	5	1	6.23	1.8	0.30	3.50	21	5.9
SA 6	1	10	0.8	10	2	6.36	59	9.35	3.50	13	3.7
SA 7	1	10	0.9	5	1	6.22	1.3	0.20	3.50	35	9.9
SA 8	1	10	0.9	10	2	6.43	94	14.68	3.50	5.4	1.6
SA 9	2	10	0.8	5	1	6.23	1.6	0.26	3.50	33	9.4
SA 10	2	10	0.8	10	2	6.23	1.2	0.20	3.50	42	12
SA 11	2	10	0.9	5	1	6.22	0.68	0.11	3.50	13	3.8
SA 12	2	10	0.9	10	2	6.22	0.96	0.15	3.50	4.5	1.3
SA 13	2	10	0.8	5	1	6.24	1.6	0.26	3.50	40	11
SA 14	2	10	0.8	10	2	6.24	1.3	0.21	3.50	14	3.9
SA 15	2	10	0.9	5	1	6.23	1.1	0.18	3.50	120	34
SA 16	2	10	0.9	10	2	6.22	0.93	0.15	3.50	7.9	2.2

**TABLE 2. Simulated Annealing Numerical Tests for GFRP Design.** *The first six columns list the name of the combination and the parameters, the next 3 columns provide the results for the minimum weight objective function, and the final 3 columns provide the results for the minimum moment of inertia objective function. The bold row indicates the most robust solution.*

	<i>LMCS Prototype (Aluminum)</i>	<i>Design 1 (GFRP)</i>	<i>Design 2 (483MPa Steel)</i>	<i>Design 3 (689MPa Steel)</i>
Modulus of Elasticity (E)	70,000 MPa	12,400 MPa	200,000 MPa	200,000 MPa
Moment of Inertia (I)	0.0016 m <sup>4</sup>	0.0035 m <sup>4</sup>	0.000217 m <sup>4</sup>	0.000220 m <sup>4</sup>
EI	112 MN	43.4 MN	43.4 MN	44.0 MN
Density ( $\rho$ )	2,800 kg/m <sup>3</sup>	1,790 kg/m <sup>3</sup>	7,850 kg/m <sup>3</sup>	7,850 kg/m <sup>3</sup>
Cross Sectional Area	0.17 m <sup>2</sup>	0.63 m <sup>2</sup>	0.12 m <sup>2</sup>	0.098 m <sup>2</sup>
Superstr. Depth (H)	0.319 m	0.222 m	0.111 m	0.118 m
Weight (W) (no hinges, floats)	446 kg/m	1,125 kg/m	942.8 kg/m	766.7 kg/m
Superstr. Weight (no floats)	884 kg/m	1,125 kg/m	942.8 kg/m	766.7 kg/m
System Weight	967 kg/m	1,208 kg/m	1,025.8 kg/m	849.7 kg/m

**TABLE 3. Comparison between Original Prototype and Proposed Designs.**

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580 2 Three configuration for causeway system: (A) stiff superstructure, (B) hinged su-  
581 perstructure, (C) re-conceptualized superstructure. Images (A) and (B) courtesy of  
582 US Army ERDC [Resio et al 2012]. . . . . 31

583 3 Hinge configurations: (A) compliant connection in original LMCS design, (B)  
584 moment connection for re-conceptualized system. Image (A) based on drawings  
585 provided by the ERDC. . . . . 32

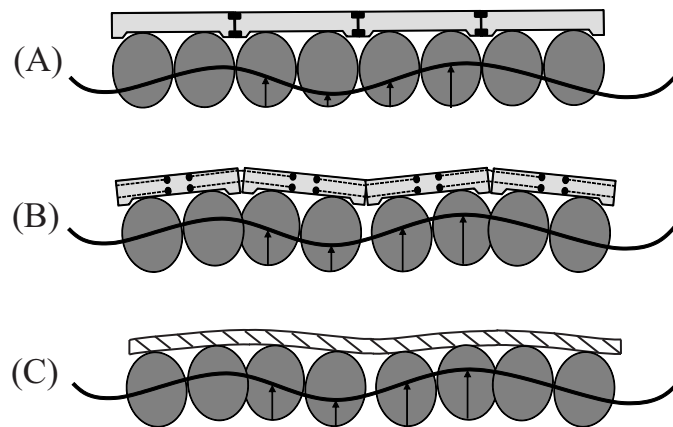
586 4 General Cross Section . . . . . 33

587 5 Pareto-Optimal (PO) Set of Solutions from One MOSA Numerical Simulation . . . 34

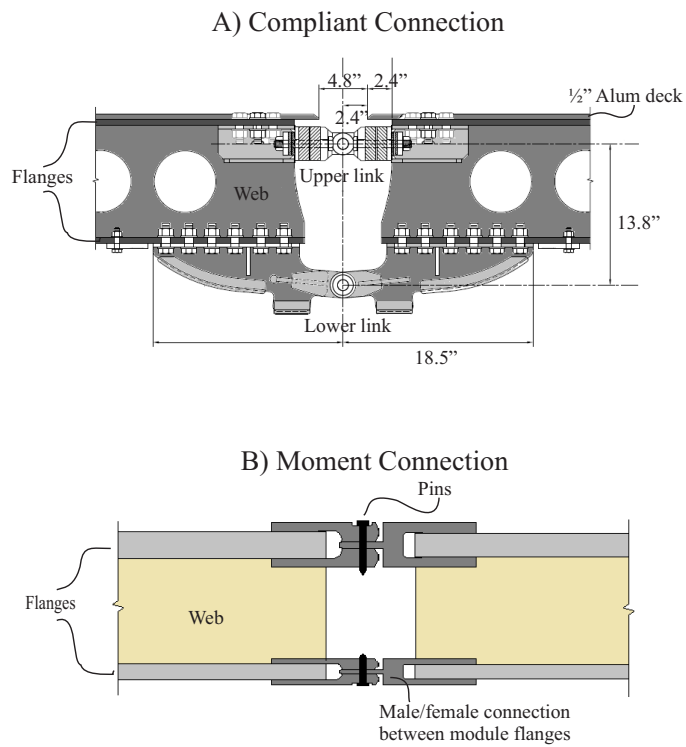
588 6 Optimized Cross Sections in GFRP (A), 483 MPa steel (B) and 689 MPa steel (C) . 35



**FIG. 1. Prototype for rapidly deployable causeway system. Image courtesy of US Army ERDC.**

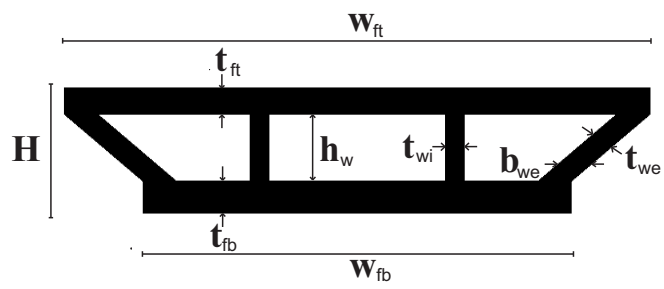


**FIG. 2. Three configuration for causeway system: (A) stiff superstructure, (B) hinged superstructure, (C) re-conceptualized superstructure. Images (A) and (B) courtesy of US Army ERDC [Resio et al 2012].**

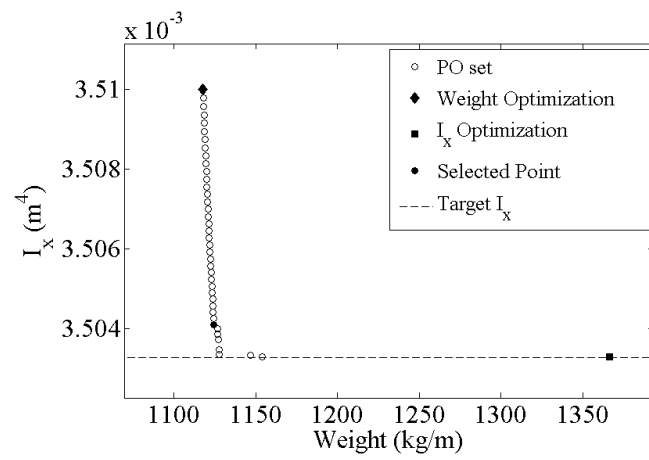


**FIG. 3. Hinge configurations: (A) compliant connection in original LMCS design, (B) moment connection for re-conceptualized system. Image (A) based on drawings provided by the ERDC.**

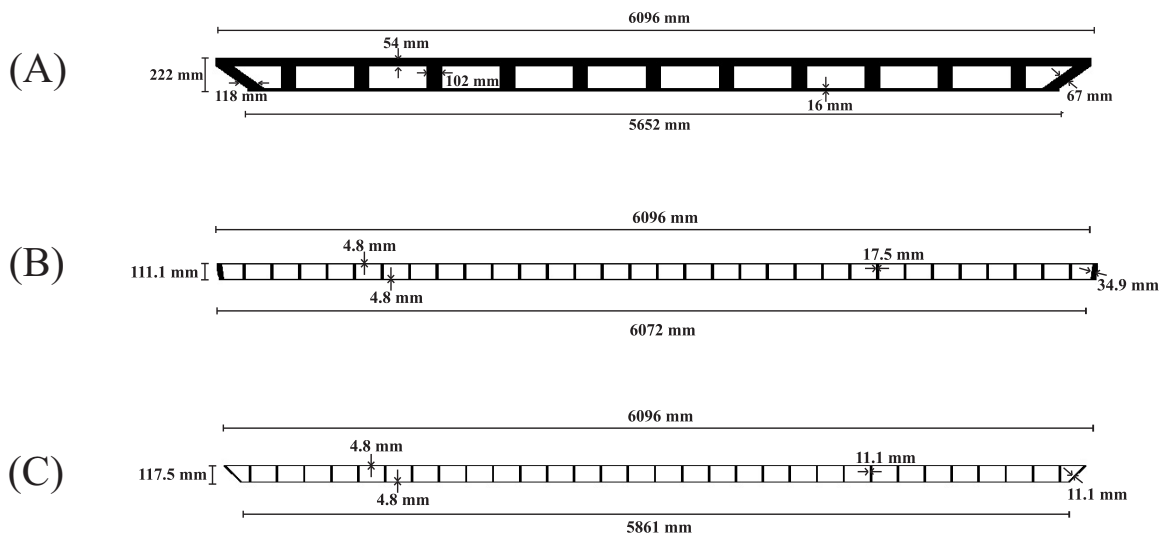




**FIG. 4. General Cross Section**



**FIG. 5. Pareto-Optimal (PO) Set of Solutions from One MOSA Numerical Simulation**



**FIG. 6. Optimized Cross Sections in GFRP (A), 483 MPa steel (B) and 689 MPa steel (C)**